

APPLICATION STUDY OF POST-TREATED HIGH-STRENGTH STEEL JOINTS IN INDUSTRIAL CRANE FRAMEWORKS

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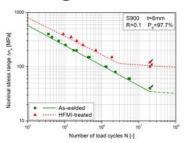


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Introduction

Experimental fatigue tests



 Evaluation of fatigue strength increase by post-treatment methods
 Statistically verified data

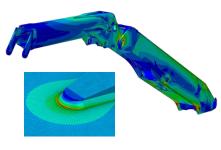
Service load measurements



$$\begin{pmatrix} F_X(t) \\ F_Y(t) \\ F_Z(t) \end{pmatrix} = Pseudoinv \begin{vmatrix} \varepsilon_{1,X} & \varepsilon_{1,Y} & \varepsilon_{1,Z} \\ \varepsilon_{2,X} & \varepsilon_{2,Y} & \varepsilon_{2,Z} \\ \varepsilon_{3,X} & \varepsilon_{3,Y} & \varepsilon_{3,Z} \end{vmatrix} \cdot \begin{pmatrix} \varepsilon_{1,measurement}(t) \\ \varepsilon_{2,measurement}(t) \\ \varepsilon_{3,measurement}(t) \end{pmatrix}$$

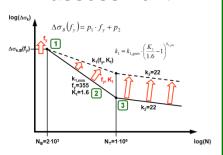
- Built-up of methodology to measure the service loads during operation
- Classification of results

Numerical analysis of structural details



 Numerical analysis of complete structure and critical details by solid, shell and submodels

Local fatigue assessment



- Local fatigue behaviour of as-welded and posttreated condition
- Lifetime assessment

Light-weight crane frameworks

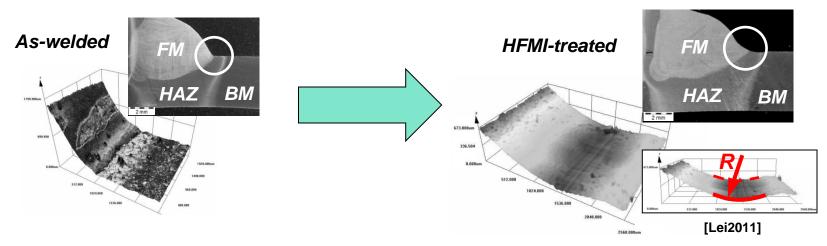




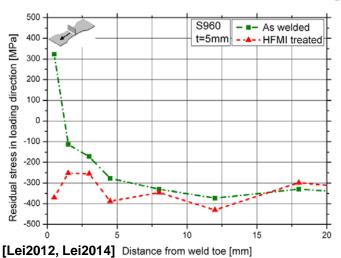


HFMI (High Frequency Mechanical Impact) as post-treatment

Improvement of notch topography



Change of residual stresses



- Surface residual stresses in transversal (load) direction (T-joint, S960)
 - As-welded: High tensile residual stresses at weld toe region (+300MPa)
 - HFMI-treated: Superposition of compressive residual stresses
 - → Reduction down to -350MPa



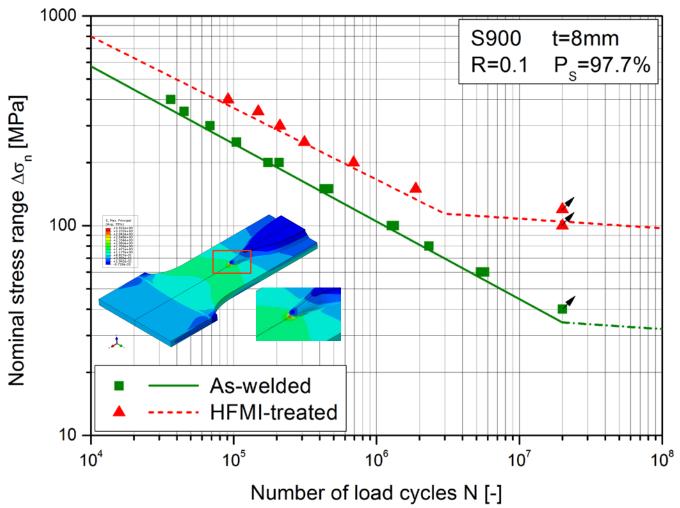
End-of-seam specimens – HFMI-treatment



- HFMI-treatment at the weld toe as post-treatment method (PIT-System)
 - HFMI-Parameters: p=6bar (~90psi), v=20-30cm/min, f=90Hz
 - Radius of the hardened pin: R=2mm



End-of-seam specimens – Nominal S/N-curves

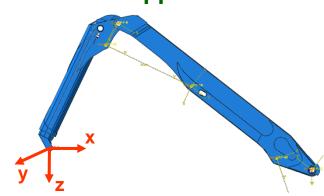


 Major increase of fatigue strength by HFMI-treatment at the weld toe due to high notch effect at weld toe region (small, highly stressed volume)



Service load measurements

 Evaluation of service loads at crane end based on pseudoinverse approach



$$F_{X} = \begin{pmatrix} 1 \\ 0 \\ 0 \end{pmatrix} \longrightarrow \mathcal{E}_{X} = \begin{pmatrix} \mathcal{E}_{1,X} \\ \mathcal{E}_{2,X} \\ \mathcal{E}_{3,X} \end{pmatrix}$$

$$F_{Y} = \begin{pmatrix} 0 \\ 1 \\ 0 \end{pmatrix} \longrightarrow \mathcal{E}_{Y} = \begin{pmatrix} \mathcal{E}_{1,Y} \\ \mathcal{E}_{2,Y} \\ \mathcal{E}_{3,Y} \end{pmatrix}$$

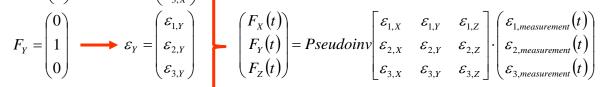
$$F_{Z} = \begin{pmatrix} 0 \\ 0 \\ 1 \end{pmatrix} \longrightarrow \mathcal{E}_{Z} = \begin{pmatrix} \mathcal{E}_{1,Z} \\ \mathcal{E}_{2,Z} \\ \mathcal{E}_{3,Z} \end{pmatrix}$$



Standard load cases F_X , F_Y and F_Z



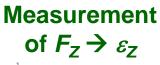


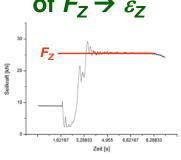


Strain gauge and cylinder displacement measurements in service

Service loads $F_X(t)$, $F_Y(t)$ and $F_Z(t)$

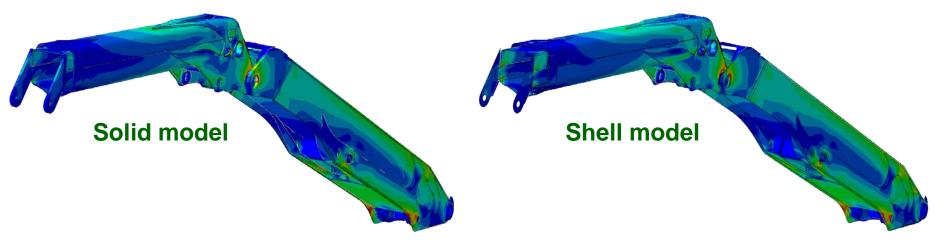




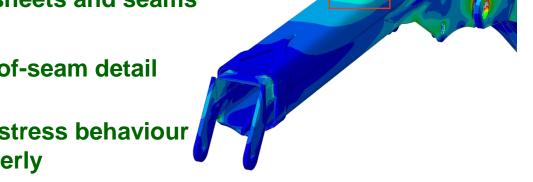




Numerical analysis – Complete model of framework



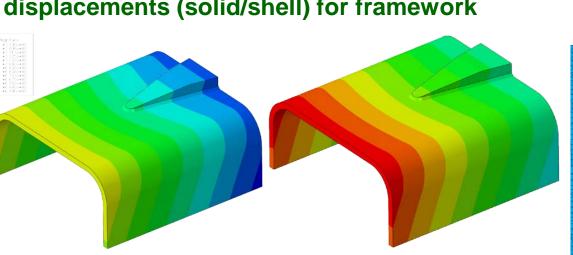
- Built-up of solid and shell model of complete framework
- Good agreement of numerical results between both modeling techniques
- Selection of simulation method depends on
 - Pre-processing effort and computational time
 - Contact modeling between sheets and seams
- Detailed analysis of critical end-of-seam detail by submodeling technique
 - Fine mesh to evaluate local stress behaviour for fatigue assessment properly





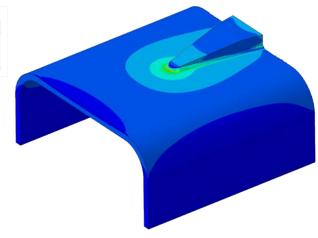
Numerical analysis – Submodel technique

- Solid submodel of specific framework detail
- Mesh according to IIW-guideline [Fri2009]
- Displacement data at cross-section points from complete solid and shell framework model
- Maximum principal stress at weld toe based on
 - Complete solid model: 110MPa
 - Complete shell model: 91MPa
- Difference caused by minor dissimilar displacements (solid/shell) for framework



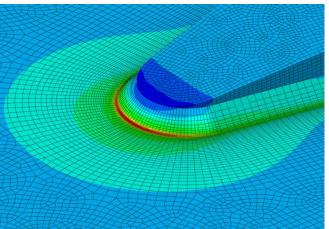
Displacement from complete solid model

Displacement from complete shell model



Displacement from complete solid model



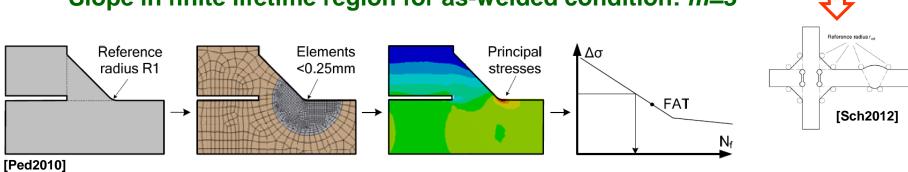


Crack at toe

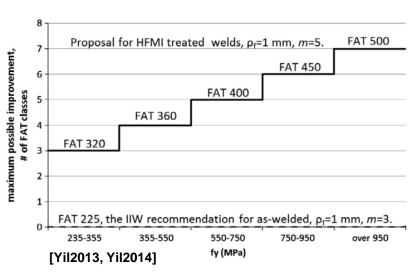


Local fatigue assessment

- Effective notch stress concept according to IIW-recommendation [Hob2009]
 - Reference radius of ρ_f =1mm based on [Neu1968, Rad1996]
 - Slope in finite lifetime region for as-welded condition: *m*=3

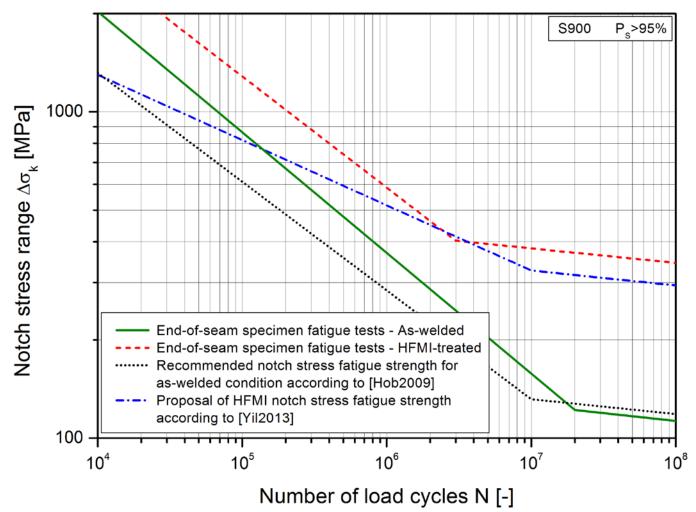


- Proposal of HFMI notch stress fatigue strength according to [Yil2013]
 - Reference radius: ρ_f =1mm
 - Slope in finite lifetime region: *m*=5
 - One fatigue class (about 12.5%) increase in fatigue strength for every 200MPa increase in ultimate base material strength f_y





Local fatigue assessment



 Notch stress fatigue assessment shows a good accordance of the fatigue tests and the recommended (as-welded) and proposed (HFMI-treated) values



Summary and Outlook

- Presentation of comprehensive methodology to assess the lifetime of crane frameworks based on experimental fatigue tests, service load measurements, numerical analyses and local fatigue behaviour
- Nominal S/N-curves of investigated HFMI-treated end-of-seam specimens show a significant beneficial increase of the fatigue strength especially in the high-cycle fatigue region up to 250% compared to the as-welded condition
- Numerical analyses involving solid, shell and submodel technique demonstrate the advantages and drawbacks of each method
- Results of the comparative local fatigue assessment are in good agreement to the proposed HFMI-treatment fatigue guideline

Outlook

- Influence of applied shell elements to study numerical effects
- Determination of stress gradient effect for HFMI-treated joints
- Enhancement of variable amplitude HFMI fatigue test data and implementation in recent guidelines [Mik2013]



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